ME:5160 (58:160) Intermediate Mechanics of Fluids Fall 2024 – HW5 Solution

P3.77 Water at 20°C flows steadily through a reducing pipe bend, as in Fig. P3.77. Known conditions are p1 = 350 kPa, D1 = 25 cm, V1 = 2.2 m/s, p2 = 120 kPa, and D2 = 8 cm. Neglecting bend and water weight, estimate the total force which must be resisted by the flange bolts.



Solution: First establish the mass flow and exit velocity:

$$\dot{m} = \rho_1 A_1 V_1 = 998 \left(\frac{\pi}{4}\right) (0.25)^2 (2.2) = 108 \ \frac{kg}{s} = 998 \left(\frac{\pi}{4}\right) (0.08)^2 V_2, \text{ or } V_2 = 21.5 \ \frac{m}{s}$$

The CV surrounds the bend and cuts through the flanges. The force balance is

$$\sum F_{x} = -F_{bolts} + p_{1,gage}A_{1} + p_{2,gage}A_{2} = \dot{m}_{2}u_{2} - \dot{m}_{1}u_{1}, \text{ where } u_{2} = -V_{2} \text{ and } u_{1} = V_{1}$$

or $F_{bolts} = (350000 - 100000)\frac{\pi}{4}(0.25)^{2} + (120000 - 100000)\frac{\pi}{4}(0.08)^{2} + 108(21.5 + 2.2)$
 $= 12271 + 101 + 2553 \approx 14900 \text{ N} \text{ Ans.}$

P3.94 A water jet 3 inches in diameter strikes a concrete (SG = 2.3) slab which rests freely on a level floor. If the slab is 1 ft wide into the paper, calculate the jet velocity which will just begin to tip the slab over.



Fig. P3.94

Solution: For water let $\rho = 1.94$ slug/ft³. Find the water force and then take moments about the lower left corner of the slab, point B. A control volume around the water flow yields

$$\sum F_x = F_{on jet} = \sum m_{out} u_{out} - \sum m_{in} u_{in} = m_{out}(0) - \rho A V(-V), \quad F = \rho A V^2$$

$$\sum M_B = (\rho A V^2) (\frac{21.5}{12} ft) - W_{slab} (\frac{4}{12} ft), \quad W_{slab} = (2.3 \times 62.4) (\frac{8}{12} ft) (3ft) (1ft) = 287 \ lbf$$

Thus $(1.94) \frac{\pi}{4} (\frac{3}{12} ft)^2 V^2 (\frac{21.5}{12} ft) = (287 lbf) (\frac{4}{12} ft), \text{ solve for } V_{jet} = 23.7 \frac{\text{ft}}{\text{s}} Ans.$

P3.153 The 3-arm lawn sprinkler of Fig. P3.153 receives 20°C water through the center at 2.7 m³/hr. If collar friction is neglected, what is the steady rotation rate in rev/min for (a) $\theta = 0^{\circ}$; (b) $\theta = 40^{\circ}$?



Solution: The velocity exiting each arm is

$$V_{o} = \frac{Q/3}{(\pi/4)d^2} = \frac{2.7/[(3600)(3)]}{(\pi/4)(0.007)^2} = 6.50 \frac{m}{s}$$

With negligible air drag and bearing friction, the steady rotation rate (Example 3.15) is

$$\omega_{\text{final}} = \frac{V_{\text{o}} \cos \theta}{R} \quad \text{(a)} \ \theta = 0^{\circ}: \quad \omega = \frac{(6.50) \cos 0^{\circ}}{0.15 \text{ m}} = 43.3 \ \frac{\text{rad}}{\text{s}} = 414 \ \frac{\text{rev}}{\text{min}} \quad Ans. \text{(a)}$$

$$\text{(b)} \ \theta = 40^{\circ}: \quad \omega = \omega_{\text{o}} \cos \theta = (414) \cos 40^{\circ} = 317 \ \frac{\text{rev}}{\text{min}} \quad Ans. \text{(b)}$$

P3.180 Water at 20°C is pumped at 1500 gal/ min from the lower to the upper reservoir, as in Fig. P3.180. Pipe friction losses are approximated by $hf \approx 27V^2/(2g)$, where V is the average velocity in the pipe. If the pump is 75 percent efficient, what horse-power is needed to drive it?



Solution: First evaluate the average velocity in the pipe and the friction head loss:

$$Q = \frac{1500}{448.8} = 3.34 \frac{\text{ft}^3}{\text{s}}$$
, so $V = \frac{Q}{A} = \frac{3.34}{\pi (3/12)^2} = 17.0 \frac{\text{ft}}{\text{s}}$ and $h_f = 27 \frac{(17.0)^2}{2(32.2)} \approx 121 \text{ ft}$

Then apply the steady flow energy equation:

$$\frac{\mathbf{p}_1}{\rho \mathbf{g}} + \frac{\mathbf{V}_1^2}{2\mathbf{g}} + \mathbf{z}_1 = \frac{\mathbf{p}_2}{\rho \mathbf{g}} + \frac{\mathbf{V}_2^2}{2\mathbf{g}} + \mathbf{z}_2 + \mathbf{h}_f - \mathbf{h}_p,$$

or: 0+0+50=0+0+150+121-h_p

Thus
$$h_p = 221$$
 ft, so $P_{pump} = \frac{\gamma Q h_p}{\eta} = \frac{(62.4)(3.34)(221)}{0.75}$

$$= 61600 \quad \frac{\text{ft} \cdot \text{lbf}}{\text{s}} \approx 112 \text{ hp} \quad Ans.$$

P3.183 The pump in Fig. P3.183 creates a 20°C water jet oriented to travel a maximum horizontal distance. System friction head losses are 6.5 m. The jet may be approximated by the trajectory of friction-less particles. What power must be delivered by the pump?



Fig. P3.183

Solution: For maximum travel, the jet must exit at $\theta = 45^{\circ}$, and the exit velocity must be

$$V_2 \sin \theta = \sqrt{2g\Delta z_{max}}$$
 or: $V_2 = \frac{[2(9.81)(25)]^{1/2}}{\sin 45^\circ} \approx 31.32 \frac{m}{s}$

The steady flow energy equation for the piping system may then be evaluated:

$$p_1/\gamma + V_1^2/2g + z_1 = p_2/\gamma + V_2^2/2g + z_2 + h_f - h_p,$$

or:
$$0+0+15=0+(31.32)^2/[2(9.81)]+2+6.5-h_p$$
, solve for $h_p \approx 43.5$ m

Then
$$P_{\text{pump}} = \gamma Q h_{\text{p}} = (9790) \left[\frac{\pi}{4} (0.05)^2 (31.32) \right] (43.5) \approx 26200 \text{ W}$$
 Ans